

# 2 POLE PLACEMENT VIA LYAPUNOV FOR CONSTRAINED CONTROL OF MISMATCHED SYSTEMS

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## 1 Introduction

The main objective discussed in [5] is the suppression of undesired noise or vibrations in dynamical systems which are modelled by an o.d.e. of the form

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}(\mathbf{x})\mathbf{u} + \mathbf{e}(\mathbf{x}, t) \quad \text{with} \quad \mathbf{x}(0) = \mathbf{x}_0 \in \mathbb{R}^n. \quad (1)$$

The variable  $\mathbf{x} \in \mathbb{R}^n$  describes the state of the system.  $\mathbf{A} \in \mathbb{R}^{n,n}$  is assumed to be a constant and stable system matrix.  $\mathbf{u} \in \mathbb{R}^m$  with  $u_k \in [-1, +1]$  for  $k = 1, \dots, m$  is the constrained control input. The input matrix  $\mathbf{B} \in C^0[\mathbb{R}^n, \mathbb{R}^m]$  may be state-dependent. The *Caratheodory function*  $\mathbf{e} : \mathbb{R}^n \times \mathbb{R} \rightarrow \mathbb{R}^n$  models time-dependence, additional nonlinearities, and unknown disturbances. However, it is assumed to be uniformly bounded; that is,  $\|\mathbf{e}(\mathbf{x}, t)\| \leq \eta$  for all  $(\mathbf{x}, t) \in \mathbb{R}^n \times \mathbb{R}$ , where the constant  $\eta \in (0, \infty)$  may be unknown.

In [5, 6] the authors showed that the control design method based on Lyapunov stability theory leads to a unique control function

$$u_k = p_k(\mathbf{x}) = -\text{sgn}[b_k(\mathbf{x})], \quad k \in \{1, \dots, m\} \quad (2)$$

that minimizes the time derivative  $\dot{V}(\mathbf{x}(t))$  of any given Lyapunov function candidate  $V \in C^1[\mathbb{R}^n, \mathbb{R}]$  along any trajectory  $t \mapsto \mathbf{x}(t)$  which satisfies equation (1) with control (2). The indicator function  $b_k$  in that case is given by

$$b_k(\mathbf{x}) = \sum_{j=1}^n \left[ \frac{\partial V}{\partial x_j}(\mathbf{x}) \cdot B_{jk}(\mathbf{x}) \right] \quad \text{where} \quad B_{jk}(\mathbf{x}) := \mathbf{e}_j^T \mathbf{B}(\mathbf{x}) \tilde{\mathbf{e}}_k. \quad (3)$$

$$(\mathbf{e}_i^T \mathbf{e}_j = \delta_{ij}, \tilde{\mathbf{e}}_i^T \tilde{\mathbf{e}}_j = \delta_{ij}, \mathbf{e}_i \in \mathbb{R}^n, \tilde{\mathbf{e}}_i \in \mathbb{R}^m).$$

As shown in [6], the commonly used Lyapunov function candidate  $V(\mathbf{x}) = \mathbf{x}^T \mathbf{P} \mathbf{x}$  with  $\mathbf{P} \in \mathbb{R}^{n,n}$  and  $\mathbf{P} > 0$  is actually a Lyapunov function if we determine  $\mathbf{P}$  via the

algebraic Lyapunov equation  $PA + A^T P = -Q$  where  $Q \in \mathbb{R}^{n,n}$  is any given positive definite matrix. In that case, the radius  $\rho$  of the ball of ultimate boundedness is given by

$$\rho = 2 \cdot \eta \cdot \frac{\lambda_{\max}(P)}{\lambda_{\min}(Q)} \cdot \sqrt{\frac{\lambda_{\max}(P)}{\lambda_{\min}(Q)}} \quad (4)$$

$\lambda_{\max}(P)$  and  $\lambda_{\min}(P)$  denote the maximum and minimum eigenvalues of  $P > 0$ . The same holds for  $Q > 0$ . At this point it should be noted that minimization of the Lyapunov derivative

$$\mathcal{L}_{(x,t)}[\mathbf{u}] := \mathbf{x}^T P (A\mathbf{x} + B\mathbf{u} + \mathbf{e}(x, t)) \quad (5)$$

has also a geometric interpretation in  $\mathbb{R}^n$ : Since  $P > 0$  there is an invertible  $T \in \mathbb{R}^{n,n}$  such that  $P = T^T T$  or, along a trajectory of (1),

$$\begin{aligned} \mathcal{L}_{(x,t)}[\mathbf{u}] &= \mathbf{x}^T T^T T \dot{\mathbf{x}} \\ &= (T\mathbf{x})^T \frac{d}{dt}(T\mathbf{x}), \quad \left[ \frac{d}{dt}(T\mathbf{x}) = T\dot{\mathbf{x}} \right] \\ &= \mathbf{y}^T \dot{\mathbf{y}}, \quad [\mathbf{y} := T\mathbf{x}]. \end{aligned} \quad (6)$$

Thus, the proposed control (2) minimizes the inner product between  $\mathbf{y}(t)$  and  $\dot{\mathbf{y}}(t)$ ; in other words,  $\dot{\mathbf{y}}(t)$  points as closely as possible towards the origin. Of course, this tendency and hence the system performance depends on the choice of  $T$  or  $P$ , respectively. In Section 3, in order to illustrate further the ‘‘optimality’’ of the proposed control (2), we employ a  $T$  such that

$$TAT^{-1} = \Omega, \quad (7)$$

where the real matrix  $\Omega$  is defined by

$$\Omega := \left[ \begin{array}{ccc|ccc} \begin{pmatrix} -\delta_1 & -\omega_1 \\ \omega_1 & -\delta_1 \end{pmatrix} & & 0 & & & \\ & \ddots & & & & 0 \\ & & \begin{pmatrix} -\delta_{n_c} & -\omega_{n_c} \\ \omega_{n_c} & -\delta_{n_c} \end{pmatrix} & & & \\ \hline & & & & -\mu_1 & 0 \\ & & 0 & & & \ddots \\ & & & & 0 & & -\mu_{n_r} \end{array} \right] \quad (8)$$

and

- $\lambda_{2k-1} = -\delta_k + i\omega_k$  and  $\lambda_{2k} = -\delta_k - i\omega_k$  are the complex eigenvalues of  $A$
- $\mu_1, \dots, \mu_{n_r}$  are the real eigenvalues of  $A$ .

Furthermore, because of

$$\begin{aligned}
 -Q &= PA + A^T P \\
 &= T^T TA + A^T T^T T \\
 &= T^T (TAT^{-1})T + T^T (T^T)^{-1} A^T T^T T \\
 &= T^T \Omega T + T^T (TAT^{-1})^T T, & (\Omega = TAT^{-1}) \\
 &= T^T (\Omega + \Omega^T) T & (9) \\
 &= -T^T \Delta T, & (\Delta := -(\Omega + \Omega^T)) \\
 &= -(\sqrt{\Delta} T)^T (\sqrt{\Delta} T) \\
 &= -R^T R, & (R := \sqrt{\Delta} T)
 \end{aligned}$$

or

$$Q = R^T R \quad (10)$$

respectively, this  $T$  defines a Lyapunov function, and (cf. [6]) the radius  $\rho$  of the ball of ultimate boundedness is given by

$$\rho = \frac{\eta}{\delta_{max}} \cdot \sqrt{\frac{\lambda_{max}(T^T T)}{\lambda_{min}(T^T T)}} \quad (11)$$

with

$$\delta_{max} := \max\{\delta_1, \dots, \delta_{n_c}, \mu_1, \dots, \mu_{n_r}\} \quad (12)$$

## 2 Lyapunov Approach as Limit Case of a Linear Constrained Control

The objective of this section is to show that for  $B(x) = B = \text{const.}$  the controller design discussed in Section 1 and in [5, 6] is a limit case of pole placement within the set of all *admissible* linear constrained feedback controllers. The attribute *admissible* denotes the restriction of  $u$  to the set of constrained linear controllers

$$\mathcal{U}_a := \{u \in C^0[\mathbb{R}^n, \mathbb{R}^m] \mid u_k(x) = -\text{sat}(\tilde{e}_k^T Kx); K \in \mathbb{R}^{m,n}; k \in \{1, \dots, m\}\} \quad (13)$$

with

$$\text{sat}(f(x)) := \begin{cases} +1 & \text{if } f(x) < -1 \\ f(x) & \text{if } -1 \leq f(x) \leq 1; \quad f \in C^0[\mathbb{R}^n, \mathbb{R}] \\ -1 & \text{if } f(x) > 1 \end{cases} \quad (14)$$

Of course, the controller  $p(x)$  described in Section 1 does not belong to  $\mathcal{U}_a$ . However, there exists a continuous parameter deformation

$$\begin{aligned} \tilde{p}_k(x, \cdot) : ]0, \frac{\pi}{2}[ &\rightarrow \mathcal{U}_a \\ \alpha &\mapsto -\text{sat}[\tan(\alpha)\tilde{e}_k^T \mathbf{B}^T \mathbf{P}x] \end{aligned} \quad (15)$$

( $k \in \{1, \dots, m\}$ ) such that

$$p_k(x) = \lim_{\alpha \rightarrow \pi/2} \tilde{p}_k(x, \alpha) \quad (16)$$

Suppose that a solution  $t \mapsto x(t)$  of (1) with  $u_k = \tilde{p}_k(x, \alpha)$  is such that there is an interval  $[t_1, t_2]$  during which not all components of  $u$  are saturated. Let

$$u = \begin{bmatrix} u_I \\ u_{II} \end{bmatrix}, \quad \mathbf{B} = [\mathbf{B}_I, \mathbf{B}_{II}] \implies \mathbf{B}u = \mathbf{B}_I u_I + \mathbf{B}_{II} u_{II} \quad (17)$$

where  $u_I$  denotes the unsaturated part of  $u$ . Then the systems behavior on  $[t_1, t_2]$  is governed by

$$\begin{aligned} \dot{x} &= \mathbf{A}x + \mathbf{B}_I u_I + \mathbf{B}_{II} u_{II} + e(x, t) \\ &= (\mathbf{A} - \tan(\alpha)\mathbf{B}_I \mathbf{B}_I^T \mathbf{P})x + \mathbf{B}_{II} u_{II} + e(x, t) \end{aligned} \quad (18)$$

That is, as in the Lyapunov approach of [5, 6], the controller design does not take care of the uncertain excitation  $e$ , but rather improves the behavior of the *nominal* system and hence its behavior in the presence of disturbances.

With respect to (17), any arbitrary but fixed  $\alpha \in ]0, \frac{\pi}{2}[$  determines a pole distribution of the *nominal* system. These poles are the eigenvalues of

$$\tilde{\mathbf{A}}(\alpha) := \mathbf{A} - \tan(\alpha)\mathbf{B}_I \mathbf{B}_I^T \mathbf{P}. \quad (19)$$

In order to investigate the damping behavior of the controlled *nominal* system, we will take a look at one of the invariants of  $\tilde{\mathbf{A}}$  which gives information about the real parts of the eigenvalues:

$$\begin{aligned} \tilde{\mathbf{A}}(\alpha) &= \text{tr}(\mathbf{A} - \tan(\alpha)\mathbf{B}_I \mathbf{B}_I^T \mathbf{P}) \\ &= \text{tr}(\mathbf{A}) - \tan(\alpha)\text{tr}(\mathbf{B}_I \mathbf{B}_I^T \mathbf{T}^T \mathbf{T}) \\ &= \text{tr}(\mathbf{A}) - \tan(\alpha)\text{tr}(\mathbf{T} \mathbf{B}_I \mathbf{B}_I^T \mathbf{T}^T) \\ &= -\sum_{k=1}^{n_c} 2\delta_k - \sum_{k=1}^{n_r} \mu_k - \tan(\alpha) \sum_{k=1}^{m_I} \sigma_k^2 \end{aligned} \quad (20)$$

where

$$\sigma_k^2 := e_k^T \mathbf{T} \mathbf{B}_I \mathbf{B}_I^T \mathbf{T}^T e_k^T. \quad (21)$$

That is, if  $\alpha \rightarrow \frac{\pi}{2}$  then

$$\begin{aligned} 1) \quad & \tilde{p}(\mathbf{x}, \alpha) \rightarrow p(\mathbf{x}) \\ 2) \quad & \sum_{k=1}^n \operatorname{Re}\{\lambda_k(A - \tan(\alpha)\mathbf{B}_i\mathbf{B}_i^T\mathbf{P})\} \rightarrow -\infty \end{aligned} \quad (22)$$

In other words, the proposed Lyapunov approach leads to the strongest possible damping “on the average”.

### 3 Test Example and Numerical Results

To illustrate the result in Section 2, we will employ a test example already used in [6], given by the system matrix  $A$

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ -\frac{1}{m_1}[k_1 + k_2] & \frac{k_2}{m_1} & -\frac{1}{m_1}[c_1 + c_2] & \frac{c_2}{m_1} \\ \frac{k_2}{m_2} & -\frac{k_2}{m_2} & \frac{c_2}{m_2} & -\frac{c_2}{m_2} \end{bmatrix} \quad (23)$$

with

$$m_i = 1[\text{kg}], \quad k_i = 1000[\text{N/m}], \quad c_i = 1[\text{Ns/m}], \quad (24)$$

and control input matrix  $B$  and excitation  $e$  given by

$$B = \begin{bmatrix} 0 \\ 0 \\ 0 \\ -\frac{1}{m_2} \end{bmatrix}, \quad e(\mathbf{x}, t) = \begin{bmatrix} 0 \\ 0 \\ -\frac{F}{m_1} \\ 0 \end{bmatrix} \quad (25)$$

with

- $F(t) = \bar{F} \sin(\nu \cdot t)$ ,
- $\bar{F} = 5[\text{N}]$ ,  $\nu \in [10[1/\text{s}], 80[1/\text{s}]]$ .

The feedback control employed is

$$\tilde{p}_k(\mathbf{x}, \alpha) = -u_{max} \cdot \operatorname{sat}[\tan(\alpha)\mathbf{x}^T\mathbf{P}\mathbf{B}\tilde{e}_k], \quad k \in \{1, \dots, 4\}. \quad (26)$$

with

- $u_{max} = 2.5[\text{N/m}]$ .

That is, we consider a fairly simple example which, however, accounts for

- mismatched uncertainty,
- constrained excitation,
- and constrained control.

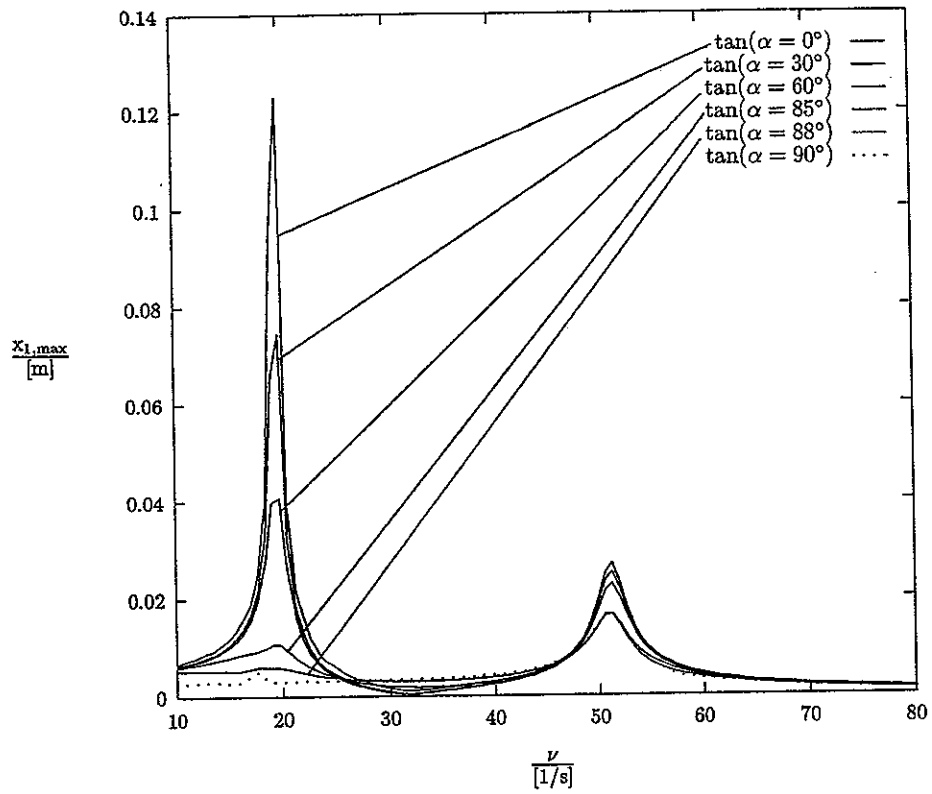


Figure 1  $x_{1, max}$  versus  $\nu$ .

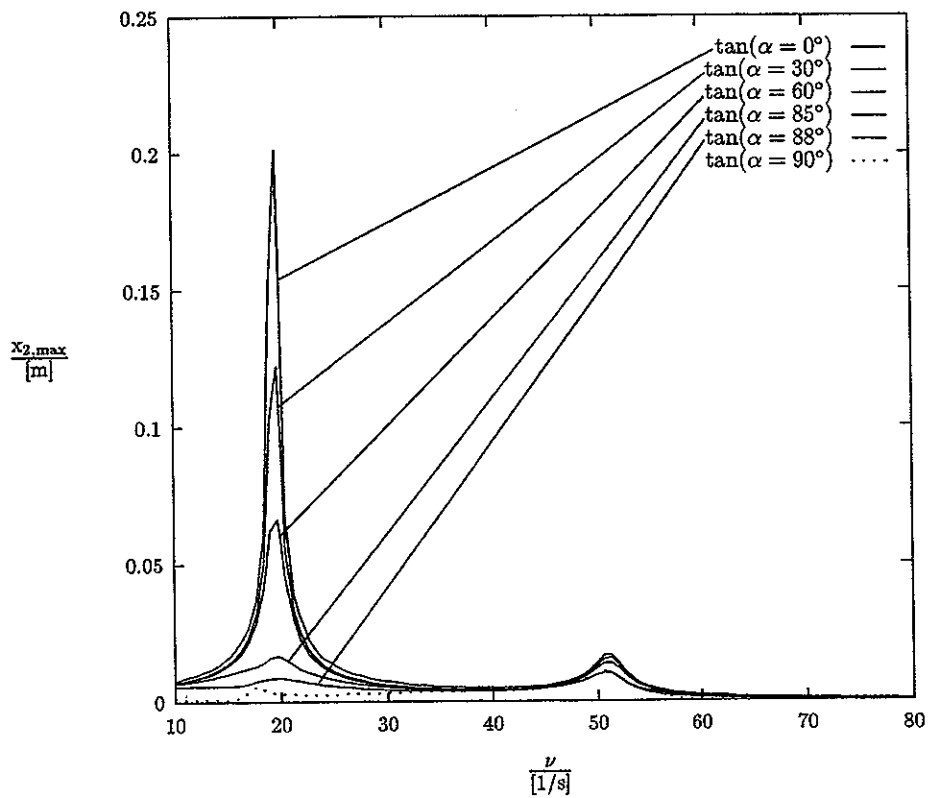


Figure 2  $x_{2, max}$  versus  $\nu$ .

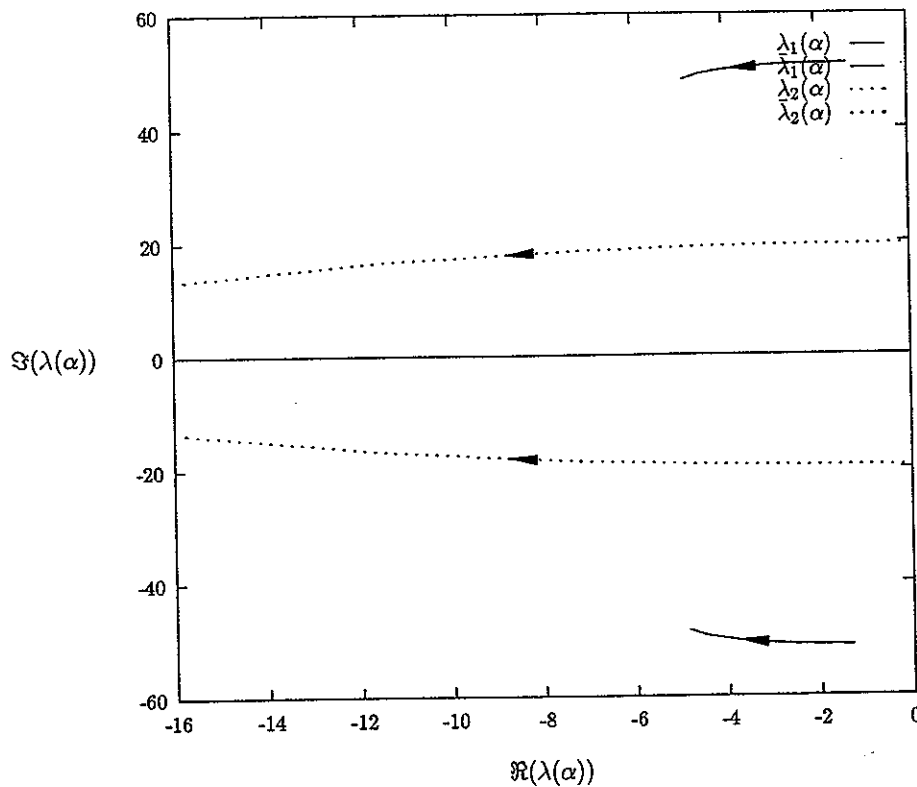


Figure 3  $\alpha \mapsto (\lambda_1, \lambda_2)$ .

Figs. 1 and 2 show the amplitudes of state variables  $x_1$  and  $x_2$  versus the excitation frequency  $\nu$  for different values of  $\alpha$ .  $\alpha = 0$  is equivalent to no control applied and  $\alpha = \pi/2$  indicates the proposed Lyapunov approach. For  $\alpha \rightarrow \pi/2$  the figures also show that vibration attenuation improves significantly.

Fig. 3 shows the eigenvalues of  $\tilde{A}(\alpha)$ . As  $\alpha$  tends to  $\pi/2$  all real parts of the eigenvalues move towards more negative values.  $Re\{\lambda_1\}$  has a strong tendency towards  $-\infty$ , while  $Re\{\lambda_2\}$  seems to move to a value less than infinity. However, on the average, the tendency towards  $-\infty$  appears to hold.

## References

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